(iii) For systems consisting of a single unit cooler or a set of multiple unit coolers serving a single piece of equipment and connected to a multiplex condensing system, by performing the following calculation:

\[
\text{Annual Energy Consumption} = \left( \frac{0.33 \times B\times H + 0.67 \times B \times L}{\text{Annual Walk-in Energy Factor}} \right) \times 8760
\]

where BLH and BLL for refrigerator and freezer systems are defined in section 7.9.2.2 and 7.9.2.3, respectively, of AHRI 1250 and the annual walk-in energy factor is calculated from the results of the test procedures set forth in AHRI 1250.


§ 431.306 Energy conservation standards and their effective dates.

(a) Each walk-in cooler or walk-in freezer manufactured on or after January 1, 2009, shall—

1. Have automatic door closers that firmly close all walk-in doors that have been closed to within 1 inch of full closure, except that this paragraph shall not apply to doors wider than 3 feet 9 inches or taller than 7 feet;

2. Have strip doors, spring hinged doors, or other method of minimizing infiltration when doors are open;

3. Contain wall, ceiling, and door insulation of at least R–25 for coolers and R–32 for freezers, except that this paragraph shall not apply to glazed portions of doors nor to structural members;

4. Contain floor insulation of at least R–28 for freezers;

5. For evaporator fan motors of under 1 horsepower and less than 460 volts, use—

   (i) Electronically commutated motors (brushless direct current motors); or
   
   (ii) 3-phase motors;

6. For condenser fan motors of under 1 horsepower, use—

   (i) Electronically commutated motors (brushless direct current motors); or
   
   (ii) Permanent split capacitor-type motors; or

   (iii) 3-phase motors; and

7. For all interior lights, use light sources with an efficacy of 40 lumens per watt or more, including ballast losses (if any), except that light sources with an efficacy of 40 lumens per watt or less, including ballast losses (if any), may be used in conjunction with a timer or device that turns off the lights within 15 minutes of when the walk-in cooler or walk-in freezer is not occupied by people.

(b) Each walk-in cooler or walk-in freezer with transparent reach-in doors manufactured on or after January 1, 2009, shall also meet the following specifications:

1. Transparent reach-in doors for walk-in freezers and windows in walk-in freezer doors shall be of triple-pane glass with either heat-reflective treated glass or gas fill.

2. Transparent reach-in doors for walk-in coolers and windows in walk-in cooler doors shall be—

   (i) Double-pane glass with heat-reflective treated glass and gas fill; or
   
   (ii) Triple-pane glass with either heat-reflective treated glass or gas fill.

3. If the walk-in cooler or walk-in freezer has an antisweat heater without antisweat heat controls, the walk-in cooler and walk-in freezer shall have a total door rail, glass, and frame heater power draw of not more than 7.1 watts per square foot of door opening (for freezers) and 3.0 watts per square foot of door opening (for coolers).

4. If the walk-in cooler or walk-in freezer has an antisweat heater with antisweat heat controls, and the total door rail, glass, and frame heater power draw is more than 7.1 watts per square foot of door opening (for freezers) and 3.0 watts per square foot of door opening (for coolers), the antisweat heat controls shall reduce the energy use of the antisweat heater.
in a quantity corresponding to the relative humidity in the air outside the door or to the condensation on the inner glass pane.

**APPENDIX A TO SUBPART R OF PART 431—UNIFORM TEST METHOD FOR THE MEASUREMENT OF ENERGY CONSUMPTION OF THE COMPONENTS OF ENVELOPES OF WALK-IN COOLERS AND WALK-IN FREEZERS**

### 1.0 Scope

This appendix covers the test requirements used to measure the energy consumption of the components that make up the envelope of a walk-in cooler or walk-in freezer.

### 2.0 Definitions

The definitions contained in § 431.302 are applicable to this appendix.

#### 3.0 Additional Definitions

**3.1 Automatic door opener/closer** means a device or control system that “automatically” opens and closes doors without direct user contact, such as a motion sensor that senses when a forklift is approaching the entrance to a door and opens it, and then closes the door after the forklift has passed.

**3.2 Core region** means the part of the panel that is not the edge region.

**3.3 Edge region** means a region of the panel that is wide enough to encompass any framing members and edge effects. If the panel contains framing members (e.g. a wood frame) then the width of the edge region must be as wide as any framing member plus 2 in. ±0.25 in. If the panel does not contain framing members then the width of the edge region must be 4 in. ±0.25 in. For walk-in panels that utilize vacuum insulated panels (VIP) for insulation, the width of the edge region must be the lesser of 4.5 in. ±1 in. or the maximum width that does not cause the VIP to be pierced by the cutting device when the edge region is cut.

**3.4 Surface area** means the area of the surface of the walk-in component that would be external to the walk-in. For example, for panel, the surface area would be the area of the side of the panel that faces the outside of the walk-in. It would not include edges of the panel that are not exposed to the outside of the walk-in.

**3.5 Rating conditions** means, unless explicitly stated otherwise, all conditions shown in Table A.1. For installations where two or more walk-in envelope components share any surface(s), the “external conditions” of the shared surface(s) must reflect the internal conditions of the adjacent walk-in. For example, if a walk-in component divides a walk-in freezer from a walk-in cooler, then the internal conditions are the freezer rating conditions and the external conditions are the cooler rating conditions.

**3.6 Percent time off (PTO)** means the percent of time that an electrical device is assumed to be off.

#### Table A.1—Temperature Conditions

<table>
<thead>
<tr>
<th>Internal Temperatures (cooled space within the envelope)</th>
<th></th>
</tr>
</thead>
<tbody>
<tr>
<td>Cooler Dry Bulb Temperature ..................................</td>
<td>35 °F.</td>
</tr>
<tr>
<td>Freezer Dry Bulb Temperature ..................................</td>
<td>−10 °F.</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>External Temperatures (space external to the envelope)</th>
<th></th>
</tr>
</thead>
<tbody>
<tr>
<td>Freezer and Cooler Dry Bulb Temperatures ...........</td>
<td>75 °F.</td>
</tr>
</tbody>
</table>

<table>
<thead>
<tr>
<th>Subfloor Temperatures</th>
<th></th>
</tr>
</thead>
<tbody>
<tr>
<td>Freezer and Cooler Dry Bulb Temperatures ...........</td>
<td>55 °F.</td>
</tr>
</tbody>
</table>

### 4.0 Calculation Instructions

**4.1 Display Panels**

(a) Calculate the U-factor of the display panel in accordance with section 5.3 of this appendix, Btu·h⁻¹·ft²·°F⁻¹.

(b) Calculate the display panel surface area, as defined in section 3.4 of this appendix, A_{dp}, ft², with standard geometric formulas or engineering software.

(c) Calculate the temperature differential, ΔT_{dp}, °F, for the display panel, as follows:

\[
\Delta T_{dp} = |T_{DB_{ext,dp}} - T_{DB_{int,dp}}| \quad (4-1)
\]

Where:

- \(T_{DB_{ext,dp}}\) = dry-bulb air external temperature, °F, as prescribed in Table A.1; and
- \(T_{DB_{int,dp}}\) = dry-bulb air temperature internal to the cooler or freezer, °F, as prescribed in Table A.1.

(d) Calculate the conduction load through the display panel, \(Q_{cond,dp}\), Btu/h, as follows: