§ 86.1333–90

 transient test cycle generation.

 (a) The heavy-duty transient engine cycles for Otto-cycle and diesel engines are listed in appendix I (f) (1), (2) and (3) to this part. These second-by-second listings represent torque and rpm maneuvers characteristic of heavy-duty engines. Both rpm and torque are normalized (expressed as a percentage of maximum) in these listings.

 (1) To unnormalize rpm, use the following equation:

 \[
 \text{Actual rpm} = \frac{\% \text{ rpm} \times (\text{Measured rated rpm} - \text{Curb idle rpm}) + \text{Curb idle rpm}}{100}
 \]

 The method of calculating measured rated rpm is detailed in paragraph (g) of this section.

 (2) Torque is normalized to the maximum torque at the rpm listed with it. Therefore, to unnormalize the torque values in the cycle, the maximum torque curve for the engine in question must be used. The generation of the maximum torque curve is described in §86.1332.

 (3) The EPA Engine Dynamometer Schedule for Heavy Duty Diesel Engines listed in appendix I (f)(2) contains torque points referred to as “closed rack motoring.” For reference cycle calculation torque points shall take on unnormalized values determined in either of the following three ways:
(i) Negative 40 percent of the positive torque available at the associated speed point. The generation of this positive maximum torque curve is described in §86.1332.

(ii) Map the amount of negative torque required to motor the engine between idle and maximum mapping speed and use this map to determine the amount of negative torque required at the associated speed point.

(iii) Determine the amount of negative torque required to motor the engine at idle and rated speeds and linearly interpolate using these two points.

(b) Example of the unnormalization procedure. The following test point shall be unnormalized:

<table>
<thead>
<tr>
<th>Percent RPM</th>
<th>Torque</th>
</tr>
</thead>
<tbody>
<tr>
<td>43</td>
<td>82</td>
</tr>
</tbody>
</table>

Given the following values:
- Measured Rated rpm = 3800.
- Curb Idle rpm = 600.

(1) Calculate actual rpm:

\[
\text{Actual rpm} = \frac{\% \text{ rpm (measured rated rpm} - \text{ curb idle rpm) + curb idle rpm}}{100}
\]

\[
= \frac{43 \times (3800 - 600)}{100} + 600
\]

\[
= \frac{43 \times 3200}{1976} + 600
\]

(2) Determine actual torque: Determine the maximum observed torque at 1976 rpm from the maximum torque curve. Then multiply this value (e.g., 358 ft-lbs) by 0.82. This results in an actual torque of 294 ft-lbs.

(c) Engine speed and torque shall be recorded at least once every second during the cold start test and hot start test. The torque and rpm feedback signals may be filtered.

(d) Idle Speed Enhancement Devices (e.g., cold idle, alternator idle, etc.). For an engine equipped with an idle speed enhancement device, the zero percent speed specified in the engine dynamometer schedules (appendix I (f)(1), (f)(2), or (f)(3) to this part) does not apply. The idle speed shall be the speed that results from the proper operation of the engine's idle speed enhancement device.

(1) During idle speed enhancement device operation, a manual transmission engine shall be allowed to idle at whatever speed is required to target a feedback torque equal to zero (using, for example, clutch disengagement, speed to torque control switching, software overrides, etc.) at those points in appendix I(f)(1), (f)(2), or (f)(3) to this part where both reference speed and reference torque are zero percent values. For each idle segment that is ten seconds or longer, the average feedback torque must be within ±10 ft-lbs of zero. To allow for transition, up to the first four seconds may be deleted from each idle segment calculation.

(2) During idle speed enhancement device operation, an automatic transmission engine shall be allowed to idle at whatever speed is required to target a feedback torque equal to CITTT (see paragraph (e)(2) of this section for definition of CITTT) at those points in appendix I(f)(1), (f)(2), or (f)(3) to this part where both reference speed and reference torque are zero percent values. For each idle segment that is ten seconds or longer, the average feedback torque must be within ±10 ft-lbs of CITTT. To allow for transition, up to the first four seconds may be deleted from each idle segment calculation.

(e) Automatic transmissions. The reference cycles in paragraphs (f) (1) and (2) of appendix I to this part shall be altered for test engines intended primarily for use with automatic transmissions.

(1) Zero percent speed for automatic transmission engines is defined as curb idle rpm (i.e., in-vehicle, coupled with automatic transmission in gear).

(2) All zero-percent speed, zero-percent torque points (idle points) shall be
modified to zero percent speed. Curb Idle Transmission Torque (CITT), except as permitted in §86.1337–90(a)(9). Also, all points with speed equal to or less than zero percent and torque less than CITT shall be modified to CITT. Motoring torque shall remain unchanged. In order to provide a smooth torque transition, all consecutive torque points that are between 0 and CITT shall be changed to CITT if the first of these is preceded or the last of these is succeeded by idle points. The manufacturer’s specified CITT shall be based upon that value observed in typical applications at the mean of the manufacturers’ specified idle speed range at stabilized temperature conditions.

(f) Clutch operation. Manual transmission engines may be tested with a clutch. If used, the clutch shall be disengaged at all zero percent speeds, zero percent torque points, but may be engaged up to two points preceding a non-zero point, and may be engaged for time segments with zero percent speed and torque points of durations less than four seconds. (See §86.1341 for allowances in the cycle validation criteria.)

(g) Measured rated rpm. The measured rated rpm corresponds to the 100 percent values specified in the reference cycles (paragraphs (f) (1) and (2) of appendix I to this part). It is generally intended to represent the rpm at which maximum brake horsepower occurs. For the purposes of this test sequence, it shall either be defined as the manufacturer’s specified rated speed, or calculated in the following way, whichever yields the higher speed:

1. From the maximum torque curve generated per §86.1332, find the maximum observed brake horsepower of the engine.
2. Calculate 98 percent of the observed maximum brake horsepower, and determine from the maximum torque curve the highest and lowest engine rpms at which this brake horsepower is observed.
3. The highest and lowest of the 98 percent power rpms represent the endpoints of an rpm range. The midpoint of this range shall be considered the measured rated rpm for cycle generation purposes.


§86.1333–2010 Transient test cycle generation.

(a) Generating transient test cycles. The heavy-duty transient engine cycles for Otto-cycle and diesel engines are listed in appendix I((f) (1), (2) and (3)) to this part. These second-by-second listings represent torque and rpm maneuvers characteristic of heavy-duty engines. Both rpm and torque are normalized (expressed as a percentage of maximum) in these listings.

1. To unnormalize rpm, use the following equations:

\[
\text{Actual rpm} = \frac{\% \text{ rpm} \cdot (\text{Max Test Speed} - \text{Curb Idle Speed}) + \text{Curb Idle Speed}}{112}
\]

Where:

\(\text{Max Test Speed}\) = the maximum test speed as calculated in 40 CFR part 1065.

(ii) For Otto-cycle engines:

\[
\text{Actual rpm} = \frac{\% \text{ rpm} \cdot (\text{Max Test Speed} - \text{Curb Idle Speed}) + \text{Curb Idle Speed}}{112}
\]

Where:

\(\text{Max Test Speed}\) = the maximum test speed as calculated in 40 CFR part 1065.