- (2) Be installed in an area away from the accommodation, service, or control space on type IG hulls;
 - (3) Be clearly marked;
- (4) Be segregated from the cargo piping by a removable spool piece in the cargo area or by at least two shut-off valves in the cargo area that have means of locking to meet §154.1870(a);
- (5) Have a means for checking for cargo vapor between the two valves under paragraph (a)(4) of this section;
- (6) Have fixed inert gas purging lines; and
- (7) Have fixed vent lines for purging with inert gas to meet § 154.1870(b).
- (b) Entrances, forced or natural ventilation intakes, exhausts, and other openings to accommodation, service, or control spaces that face the bow or stern loading area must meet §154.330.

§ 154.356 Cargo emergency jettisoning piping.

Emergency jettisoning piping must:

- (a) Meet §154.355(a):
- (b) Be designed to allow cargo discharge without the outer hull steel temperature falling below the minimum temperatures under §§ 154.170 and 154.172; and
- (c) Be specially approved by the Commandant (CG-522).

[CGD 74–289, 44 FR 26009, May 3, 1979, as amended by CGD 82–063b, 48 FR 4782, Feb. 3, 1983]

CARGO CONTAINMENT SYSTEMS

§154.401 Definitions.

As used in §§ 154.440 and 154.447:

" σ_Y " means the minimum yield strength of the tank material, including weld metal, at room temperature.

 $^{\prime\prime}\sigma_B^{\prime\prime}$ means minimum tensile strength of the tank material, including weld metals, at room temperature.

$\$\,154.405~$ Design vapor pressure (P_{o}) of a cargo tank.

- (a) The design vapor pressure (P_{o}) of a cargo tank must be equal to or greater than the MARVS.
- (b) The $P_{\rm o}$ of a cargo tank must be equal to or greater than the vapor pressure of the cargo at 45 °C (113 °F) if:
- (1) The cargo tank has no temperature control for the cargo; and

- (2) The vapor pressure of the cargo results solely from ambient temperature.
- (c) The $P_{\rm o}$ of a cargo tank may be exceeded under harbor conditions if specially approved by the Commandant (CG-522).

[CGD 74-289, 44 FR 26009, May 3, 1979, as amended by CGD 82-063b, 48 FR 4782, Feb. 3, 1983]

§ 154.406 Design loads for cargo tanks and fixtures: General.

- (a) Calculations must show that a cargo tank and its fixtures are designed for the following loads:
 - (1) Internal pressure head.
 - (2) External pressure load.
- (3) Dynamic loads resulting from the motion of the vessel.
- (4) Transient or stationary thermal loads if the design temperature is colder that -55 °C (-67 °F) or causes thermal stresses in cargo tank supports.
- (5) Sloshing loads, if the cargo tank is designed for partial loads.
- (6) Loads resulting from vessel's deflection.
- (7) Tank weight, cargo weight, and corresponding support reaction.
 - (8) Insulation weight.
- (9) Loads of a pipe tower and any other attachments to the cargo tank.
- (10) Vapor pressure loads in harbor conditions allowed under §154.405.
- (11) Gas pressurization if the cargo tank is designed for gas pressurization as a means of cargo transfer.
- (b) A cargo tank must be designed for the most unfavorable static heel angle within a 0° to 30° range without exceeding the allowable stress of the material.
- (c) A hydrostatic or hydropneumatic test design load must be specially approved by the Commandant (CG-522).

[CGD 74-289, 44 FR 26009, May 3, 1979, as amended by CGD 82-063b, 48 FR 4782, Feb. 3, 1983]

§ 154.407 Cargo tank internal pressure head.

(a) For the calculation required under $\S154.406(a)(1)$ and (b), the internal pressure head (h_{eq}) , must be determined from the following formula:

 $h_{eq}=10 P_o+(h_{gd})_{max}$

where: